

## About a Torsional Problem for the Orthotropic Non-Homogeneous Rod of Rectangular Cross-Section

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The paper deals with an analytical solution of the torsional problem for the orthotropic non-homogeneous rod of rectangular cross-section with two shear moduli which are functions of the cross-sectional height coordinate. Computational results are presented for the torsional rigidity, maximum shear stresses and their locations for various values of the problem parameters.

The solution of a torsional problem for the orthotropic non-homogeneous rod of rectangular cross section (Figure 1) with variable shear moduli under the action of a torsional moment  $M$  is considered. It is supposed that the planes of elastic symmetry are parallel to the sides of the rod, and two shear moduli  $G_1$  and  $G_2$  are given in the following form

$$G_1(y) = \beta^{-2} G_2(y) \quad G_2(y) = g \left( e^{-\alpha y} + C e^{\alpha y} \right)^2 \quad (1)$$

where  $\beta$ ,  $g$ ,  $\alpha$ ,  $C$  are constants. Giving various values to the parameter  $C$  it is possible to simulate both hardening and unhardening of various zones of the rod material. When  $|C| > e^{2\alpha b}$  and  $|C| < e^{-2\alpha b}$  the monotonic variation of moduli takes place.

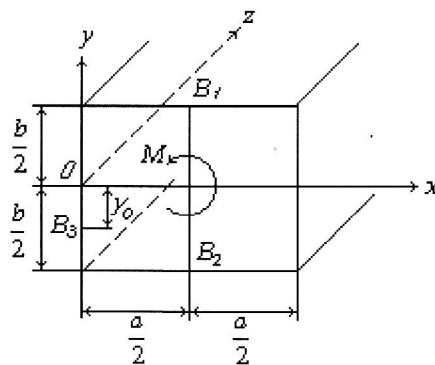


Figure 1. Non-Homogeneous Rod of Rectangular Cross-Section

Hereafter for definiteness we assume that  $\alpha=1/b$ . It should be noted that the case  $C=1$  was considered by Eishinskii et al. (1973).

Plots of the value  $G_2(y)$  depending on  $C$  are presented in Figure 2.

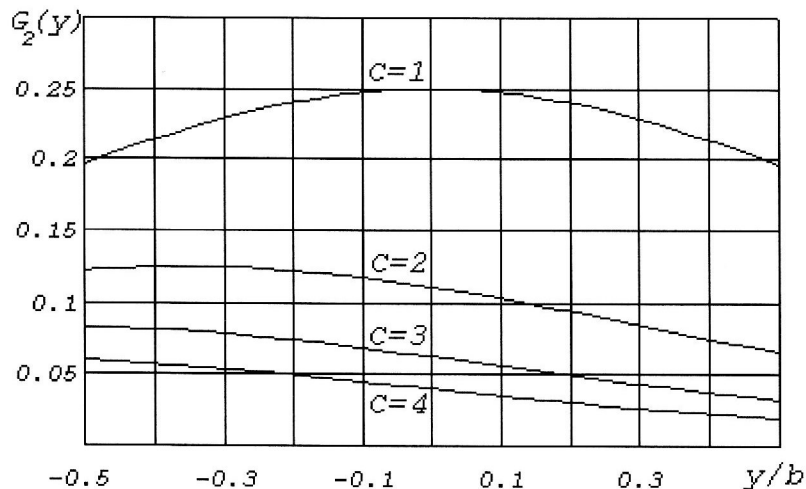


Figure 2. Plots of the Value  $G_2(y)$  Depending on  $C$

The equation for the stresses function  $\psi(x,y)$  has the form (see Lechnicky, 1972)

$$\frac{\partial}{\partial x} \left[ \frac{1}{G_1(y)} \frac{\partial \psi}{\partial x} \right] + \frac{\partial}{\partial y} \left[ \frac{1}{G_2(y)} \frac{\partial \psi}{\partial y} \right] = -2 \quad (2)$$

Solving equation (2), in view of relations (1), by the method of expansion in Fourier series, we obtain

$$\psi(x,y) = \sum_{k=1,3,5,\dots}^{\infty} \sin \frac{k\pi x}{a} Y_k(y) \quad (3)$$

where

$$Y_k(y) = \frac{4gb^2}{\pi k s_k \left( e^{-\frac{y}{b}} + C e^{\frac{y}{b}} \right)} \left\{ \frac{e^{-s_k \frac{y}{b}}}{e^{-s_k} - e^{s_k}} \left[ \int_0^{0,5} \frac{e^{-s_k t} dt}{e^{-t} + C e^t} - e^{-s_k} \int_0^{0,5} \frac{e^{s_k t} dt}{e^{-t} + C e^t} - \int_0^{-0,5} \frac{e^{-s_k t} dt}{e^{-t} + C e^t} + \right. \right.$$

$$+ e^{s_k} \int_0^{-0,5} \frac{e^{s_k t} dt}{e^{-t} + C e^t} \left. \right] + \frac{e^{s_k \frac{y}{b}}}{e^{-s_k} - e^{s_k}} \left[ e^{-s_k} \int_0^{-0,5} \frac{e^{-s_k t} dt}{e^{-t} + C e^t} - \int_0^{-0,5} \frac{e^{s_k t} dt}{e^{-t} + C e^t} - \right.$$

$$\left. - e^{s_k} \int_0^{0,5} \frac{e^{-s_k t} dt}{e^{-t} + C e^t} + \int_0^{0,5} \frac{e^{s_k t} dt}{e^{-t} + C e^t} \right] + e^{-s_k \frac{y}{b}} \int_0^{\frac{y}{b}} \frac{e^{s_k t} dt}{e^{-t} + C e^t} - e^{s_k \frac{y}{b}} \int_0^{\frac{y}{b}} \frac{e^{-s_k t} dt}{e^{-t} + C e^t} \left. \right\}$$

$$s_k = \sqrt{\left( \frac{k\pi}{d} \right)^2 + 1}$$

$$d = \frac{a}{b\beta}$$

The series (3) is absolutely convergent because the remainder after the  $k$ -th term is  $O(k^{-3})$ . Its first derivatives with respect to  $x$  and  $y$  are absolutely convergent too. In case of  $C=0$  solution (3) coincides with the well-known Lechnicky's solution (Lechnicky, 1971).

The stress state of the rod can be determined as follows

$$\tau_{xz} = \vartheta \frac{\partial \psi}{\partial y} \quad \tau_{yz} = -\vartheta \frac{\partial \psi}{\partial x}$$

where

$$\vartheta = \frac{M}{D} \text{ is the torsion angle per unit length;}$$

$$D = 2 \int_{-\frac{b}{2}}^{\frac{b}{2}} \int_0^a \psi(x,y) dx dy = gab^3 B \text{ is the torsional rigidity.}$$

The values of the factor  $B$  are plotted in Figure 3 for various values of  $C$  and  $d$ . Maximum shear stresses are determined using the following formulas:

$$\text{At point } B_1\left(\frac{a}{2}, \frac{b}{2}\right): \quad \tau_{\max} = \frac{M}{ab^2} k_1(B_1)$$

$$\text{At point } B_2\left(\frac{a}{2}, -\frac{b}{2}\right): \quad \tau_{\max} = \frac{M}{ab^2} k_1(B_2)$$

$$\text{At point } B_3(0, y_0): \quad \tau_{\max} = \frac{M}{ab^2\beta} k_2(B_3)$$

where

$$k_1(B_1) = \sum_{k=1,3,5,\dots}^{\infty} (-1)^{\frac{k-1}{2}} Y'_k\left(\frac{b}{2}\right)$$

$$k_1(B_2) = \sum_{k=1,3,5,\dots}^{\infty} (-1)^{\frac{k-1}{2}} Y'_k\left(-\frac{b}{2}\right)$$

$$k_2(B_3) = \sum_{k=1,3,5,\dots}^{\infty} k Y_k(y_0) \quad \sum_{k=1,3,5,\dots}^{\infty} k Y'_k(y_0) = 0$$

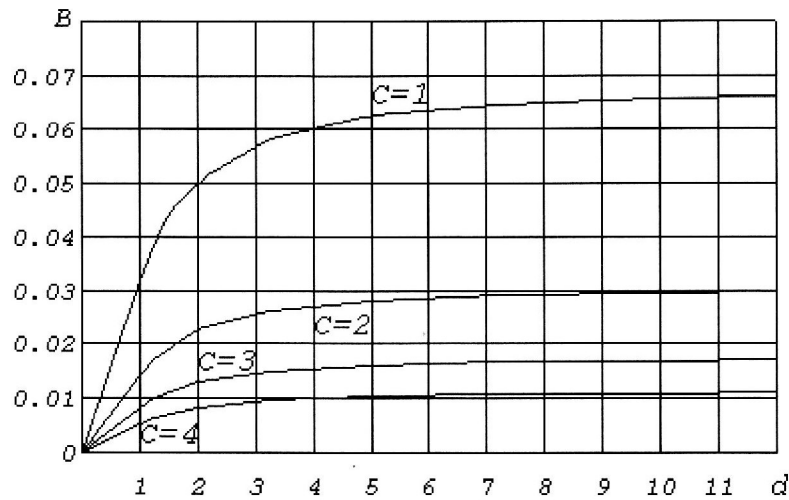


Figure 3. The Values of the Factor  $B$  Depending on  $C$  and  $d$

The values of  $k_1(B_1)$ ,  $k_1(B_2)$  depending on  $C$  and  $d$  are presented in Figure 4. It should be noted that the quantity  $k_2(B_3)$  has only a weak dependency on  $C$ , therefore its values are presented not in a graphical but in a tabulated form (see Table 1).

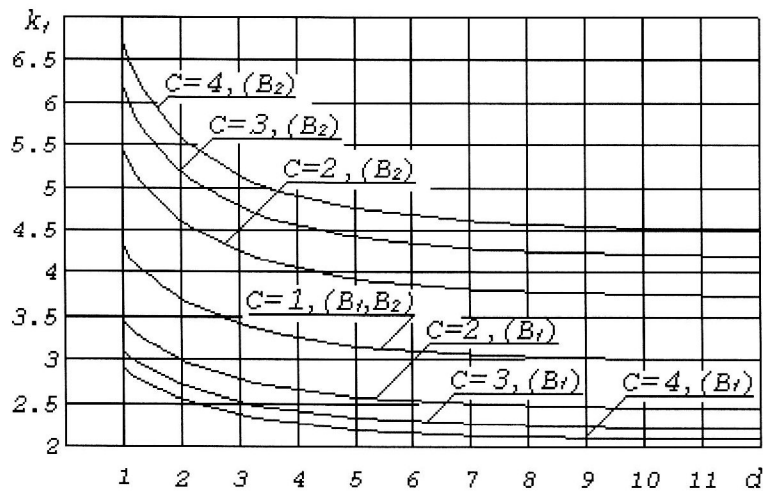


Figure 4. The Values of  $k_1(B_1)$ ,  $k_1(B_2)$  Depending on  $C$  and  $d$

The values of  $y_0/b$  are presented in Figure 5. It is evident that for  $C=1, y_0=0$ .

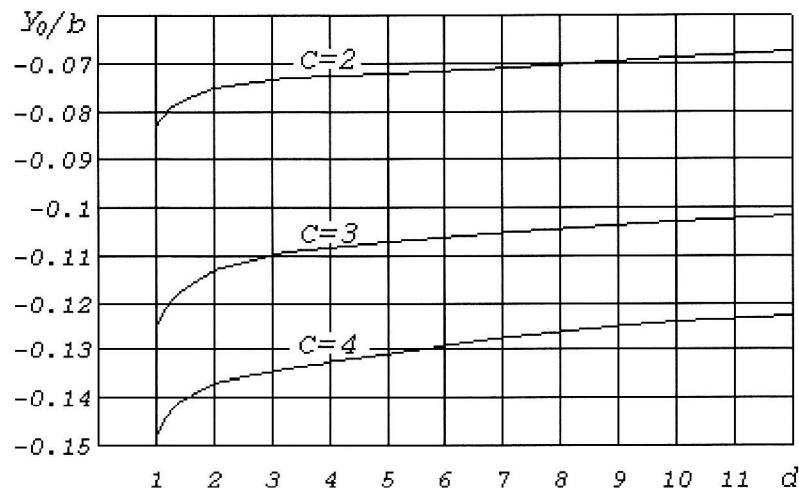


Figure 5. The Values of  $y_0/b$  Depending on  $C$  and  $d$

| $d$   | $C=2$ | $C=3$ | $C=4$ |
|-------|-------|-------|-------|
| 1.00  | 4.87  | 4.95  | 5.01  |
| 1.25  | 4.17  | 4.23  | 4.32  |
| 2.00  | 3.25  | 3.30  | 3.33  |
| 3.25  | 2.75  | 2.78  | 2.84  |
| 5.00  | 2.51  | 2.54  | 2.57  |
| 7.25  | 2.37  | 2.40  | 2.43  |
| 10.00 | 2.27  | 2.30  | 2.32  |
| 13.25 | 2.19  | 2.22  | 2.24  |

Table 1. The Values of  $k_2(B_3)$  Depending on  $C$  and  $d$

It follows from the data above that the material non-homogeneity has essential influence on the distribution of the stresses in the rod.

Example:

Let us assume that  $\beta = 1, a/b = 2, C = 3$ . Then  $d=2$ . With the results from Figures 4,5 and Table 1 we can obtain that

$$\tau_{\max}(B_1) = 2.70 \frac{M}{ab^2}, \tau_{\max}(B_2) = 5.30 \frac{M}{ab^2}, \tau_{\max}(B_3) = 3.00 \frac{M}{ab^2}, y_0 = -0.113 b.$$

### Literature

1. Eishinskii, A. M.; Adlucky, V. J.; Tsadikova E. T.; et al.: Torsion of the non-homogeneous rods with cross sections in the form of a rectangular and part of a circular ring (Russian). Modern problems of machinery industry and details of machines. Dnepropetrovsk, (1973), 132-149.
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